TECHNOLOGY

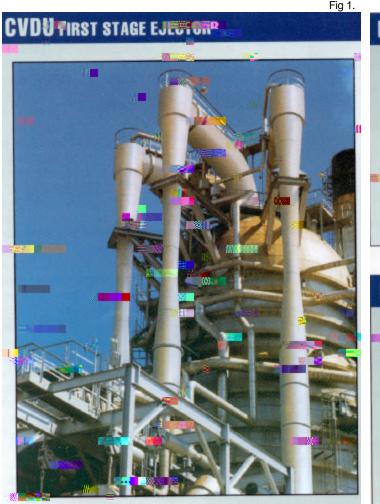
Understanding ejector systems necessary to troubleshoot vacuum distillation

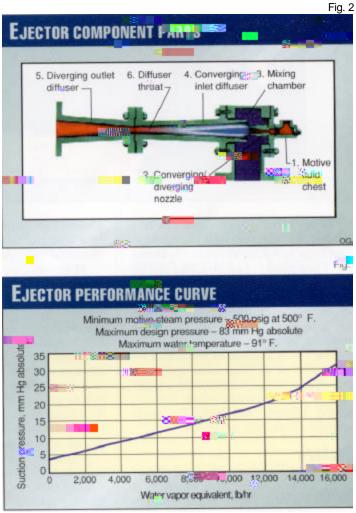
James R. Lines Graham Corp. Batavia, NY

A complete understanding of ejector system performance characteristics can reduce the time and expense associated with troubleshooting poor crude

vacuum	distillation	unit	(CVDU)
performa	nce.		
Variables	that may	negatively	/ impact
the ejec	tor-system	performa	ance of
vacuum-c	rude distilla	tion units	include

utilities supply, corrosion and erosion, fouling, and process conditions.





Tables 1 and 2 are troubleshooting guides to ejector and condenser problems in vacuum ejector systems. Fig. 1 is a photo of an installed ejector at a CVDU.

Two actual case studies conducted by service engineers on CVDU-ejector systems show how to troubleshoot ejector problems. The first problem was a result of improper replacement of an intercondenser, and the second was a result of underestimation of noncondensible loading during design, which has recently become a common problem.

Ejectors

An ejector converts pressure energy of motive steam into velocity. It has no moving

parts. Major components of an ejector consist of the

motive nozzle, motive chest, suction chamber, and diffuser (Fig. 2).

High velocity is achieved through adiabatic expansion of motive steam across a convergent/divergent steam nozzle. This expansion of steam from the motive pressure to the suction fluid operating pressure results in supersonic velocities at the exit of the

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E	SECTOR TROUBLESHOOTING		
	Problem	Effect	Correction
1	Lower than design motive-steam pressure.	Poor ejector performance.	Raise steam pressure or bore steam nozzles
2	Higher than design motive-steam pressure.	Reduced ejector capacity and wast. 21.98mm	Reduce motive pressure of Uplace steam nozzles with new nozzles designed for a high er steam pressure.
3	Higher than design steam temperature (50° F.+).	Poor ejector performance.	Raise eam pressure or bore steam nozzles.
4	Higher than design discharge pressure.	Poor ejector performance.	Look downstream of problems: a. Cooptenser problem b. Downstream ejector toblem of Sischar and provide or triction.
5	Low enderedischarge temperature. Ejector- discharge temper granuhould be superheated at least 50° F, above saturation. When S cause is wet motive steam.	Reduced ejector car now or poor performance.	a. Insulate steam lines b. Add moisture separator in s team ie.
5	Higher-than-design suction pressure (assuming reaction) steam pressure and quality are normal and discharge pressure is equal to or less than design)	Greater than design load or mechanical problems with the second Either worn out inter 0 or possible internal steam leak around steam-nozzle threads.	Inspect internal dimensions and replace to necessary. Tighten steam noizile to steam chest if necessary. Tighten steam noizile to steam chest.
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C	OND FR. TROUBLESHOOTING		
*	Problem	Effect	Corrective action
1	High ΔP across shellside (As a rule of thumb, normally ΔP should be 5% of absolute design operating pressure or less).	Poor condenser performance: a. Shell sid. If followed touling b. County water temperature higher than design c. Low cooling water flowrate d. Higher-than-design condensible hydrocarbon (about 20-30% above design).	a. Clean tubes b. Reduce temperature, increase water flow c. Increase cooling water flow d. Reduce hydrocarbon load or larger condenser and downstream elector required.
2	High _{eff} than d <mark>er igg u</mark> beside ΔP.	Poor condenser performance: a. Tubeside fouling b. Higher-than-design cooling water flow.	atter mit tubes
3	Higher than grean tubeside aT.	Poor condersor performance: a. Low cooling water flow b. Higher than design target	a. Increase flowrate b. Increase cooling water flowrate or replace condenser.
1	Hinh vanor-outlet temperature	Poor condenser performance	

High vapor-outlet temperature oor condenser performance. a Tube foulis a. Cicles tubes b. Cooling water flowrate low or inlet temperature high b. Increase water flowrate or reduce inlet temperature Possible otornal bypassing. Descent and elector not functioning and C. Check v fanturer d. D d. Check with manufacturer backstreaming I

Table 3

temperature is appreciably above the design value, insufficient steam passes through the motive nozzle. Both lowerthan-design steam pressure and higherthan-design steam temperature increase the specific volume of the motive steam and reduces the amount of steam through a motive nozzle.

In certain cases, it is possible to re-bore an ejector-motive nozzle to permit the passage of more steam through the nozzle, thereby increasing the energy available to entrain and compress the suction load.

If motive-steam pressure is more than 20% above design, too much steam expands across the nozzle. This often chokes the diffuser throat of an ejector. When this occurs, less suction load is handled by an ejector, and the CVD-column pressure rises. If an increase in column pressure is undesirable, then

	_X Design			
	Flow rate, lb/hr		Acta Flow rate, lb/hr	and the second se
Noncondensible gas	700	40	1,500	32
Water por	13,000	18	15,000	18
Condensible hydrocarbon	7,500	170	13.000	170

new ejector nozzles with smaller throat diameters are required.

Steam quality

Wet steam is very damaging to an ejector system because high-velocity moisture droplets are erosive. These droplets are rapidly accelerated as steam expands across a motive nozzle.

Erosion of nozzle internals caused by wet motive-steam is noticeable when inspecting ejector nozzles or diffuser internals. There is an etched striated pattern on the diverging section of a motive nozzle, and the nozzle mouth may actually wear out. Also, the inlet diffuser section of an ejector will show signs of erosion as a result of direct impingement of moisture droplets (Fig. 4a).

Fig. 4b depicts an ejector cutaway showing severe damage caused by wet steam. The inlet diffuser shows

substantial metal loss. Metal-scale buildup can be seen in the outlet diffuser section. The exhaust temperature from the ejector can determine if the steam conditions are present. Typical ejector exhaust temperatures are in the range of 250 to 300° F. If moisture is present, a substantially lower exhaust temperature will exist.

To solve wet-steam problems, all lines up to an ejector should be well insulated. A steam separator and trap should be installed immediately before the motive-steam inlet connection of each ejector. In some instances, a steam superheater may be required.

Wet steam can also cause performance problems. Moisture droplets through an ejector nozzle decrease the energy available for compression. This reduces the suctionload handling capacity of an ejector.

Also

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The ejector following a condenser may not be able to handle increased loading at that operating pressure of the condenser. The ejector preceding that condenser is unable to compress to a higher discharge pressure. This discontinuity in pressure causes the preceding ejector to break operation. When actual noncondensible loading is consistently above design, new ejectors